AMENDMENTS TO THE SPECIFICATION:

On page 2, please replace paragraph [0005] with the following amended paragraph:

[0005] This object is attained in that the fluid conduit is embodied such that a **swirl-type** rotation ~~**(swirl)**~~ about the longitudinal axis of the fluid conduit is impressed on the fluid stream that flows toward the valve chamber.

On page 2, please replace paragraph [0007] with the following amended paragraph:

[0007] The **swirl-type** rotation ~~**("swirl" or "spin")**~~ impressed on the flow leads to centrifugal forces, by which the flow is pressed against the wall. In this way, the fluid stream is prevented from detaching from the wall of the fluid conduit, for instance in the event of a change of direction causing the formation of a low-pressure zone. As a result, the dynamic pressure in the deflection region is reduced, and the flow resistance is lowered. Cavitation damage in the fluid conduit is also avoided. Because of the fluid stream pressing against the wall of the fluid conduit, the fluid conduit is filled uniformly, which leads to higher throughput for the same opening duration of the valve element.

On page 3, please replace paragraph [0011] with the following amended paragraph:

[0011] It is proposed that the fluid conduit includes a first conduit portion and adjoining it a second conduit portion, and the longitudinal axes of the conduit portions are at an angle $< 180°$ to one another, and the longitudinal axis of the first conduit portion is laterally offset

from the longitudinal axis of the second conduit portion. As a result of the lateral offset, the **swirl-type** rotation of the flow in the second conduit portion is brought about in a simple **manner**. Turbulence caused by the kink between the two conduit portions is effectively smoothed, or such turbulence cannot even arise in the first place.

On pages 3-4, please replace paragraph [0012] with the following amended paragraph:

[0012] The **swirl-type** rotation is especially pronounced whenever the longitudinal axes of the two conduit portions are at least approximately at a right angle to one another. In this case, the spin impressed on the flow in the second conduit portion is the greatest, and the advantages attainable with the valve assembly of the invention are therefore also the greatest.

On page 4, please replace paragraph [0013] with the following amended paragraph:

[0013] It is also proposed that the valve assembly includes a ball or a cone element as the valve element. Because of the **swirl-type** rotary motion of the fluid flowing to the valve chamber, these rotationally symmetrical valve elements are also set into rotation. This prevents unilateral wear of these valve elements and increases the durability of a valve seat associated with the valve element.

On pages 9-10, please replace paragraph [0040] with the following amended paragraph:

[0040] An alternative embodiment is shown in Fig. 8. Those elements and regions that have equivalent functions to elements and regions of the previous drawings are identified by the same reference numerals. They are not described again in detail. In contrast to the first exemplary embodiment, here the longitudinal axis 38 of the first conduit portion 34 is not at an angle of 90° but rather at an angle of approximately 45° to the longitudinal axis 40 of the second conduit portion 36. As a result, **the first and second conduit portions 34 and 36 are at an angle of approximately 135°, and thus** a more-favorable flow, or in other words a flow with less resistance, is additionally achieved.